Module of the KLIPPEL ANALYZER SYSTEM (Document Revision 1.5)

PRELIMINARY SPECIFICATION

This specification is preliminary and is subject to change.

FEATURES

- Linear signal modeling from digital input to acoustical output.
- Lumped network parameters for passive components
- Automatic equalization (DSP)
- Small signal performance for any audio input (music, test signal)
- Efficiency and voltage sensitivity versus frequency and broadband signals

BENEFITS

- Small signal performance in target application
- Considers digital, electrical, mechanical, acoustical components
- Minimum set of essential parameters
- Fast calculation of frequency responses
- Filter parameters for optimal system alignment
- Basis for large signal modeling (SIM)

DESCRIPTION

The *LSIM Linear Simulation* describes an active loudspeaker or headphone driver by using a linear lumped parameter model. Main components are equalizer, amplifier, transducer and enclosure. Using any selected input spectrum (e.g. music), meaningful statistical single values (e.g. mean efficiency) and various state spectra (e.g. SPL) are calculated. This is a useful base for defining transducer and amplifier requirements and providing significant information about the audio performance. Various transfer functions reveal the relationship between digital, electrical, mechanical and acoustical signals.

The *LSIM* features an easy-to-use simulation software with lumped or geometrical input parameters for initial (small signal) design, which is the basis for the large signal simulation in other Klippel software modules (*SIM Simulation*, *SIM-AUR Auralization*).

| Article number | 1000-300 |
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1 Overview

1.1 Principle

Basic

Principle

The LSIM Linear Simulation module illustrates a simplified linear active loudspeaker containing a band pass filter section for simulating a crossover, a prefilter (Equalization) specified by the transfer function $H_{equ}(f)$, an amplifier with an output resistance of R_g and an electrodynamical transducer mounted in an enclosure. The optimal equalizer transfer function $H_{equ}(f)$ for system alignment will be calculated automatically for a specified target transfer behaviour.

Signal based system design is possible by defining a relative input spectrum $G_w(f)$. Pink noise, typical program material according to *IEC 60268-21* and an option for individual external stimulus are provided. All spectra are converted into third octave spaced spectra. Based on this, state variables like U_g (amplifier output voltage without load) or U_T (terminal voltage) and further characteristics like SPL_{max} can be predicted. Entering a crest factor provides the option to estimate peak values.



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| Lumped Parameter Model | <i>u</i> _g — | $H_{\nu}(f) \rightarrow v$ | H _a (| (f) | $q_a \frac{H_{rad}}{d}$ | $(f) \rightarrow p(\mathbf{r})$ |
|------------------------------|--|----------------------------|--|-----------------------------------|-------------------------|---------------------------------|
| | $ \begin{array}{c} i\\ R_{G}\\ u\\ u_{g}\\ \end{array} $ | A_M F_L | $\begin{array}{c c} & & & \\ & & & & \\ & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & &$ | | q_a Z_{AL} – | $\rightarrow P_a$ |
| | amplifier | motor, exciter | diaphragm | enclosure, passive radiator | air Ioad | sound field |
| | The ISIM Linear Simulation | exciter | | radiator | load | TIEIO |

The LSIM Linear Simulation module uses a lumped-parameter model of an electro-dynamical transducer mounted in common enclosures. This model is based on chain matrices describing the different parts of the loudspeaker. A_M describes the motor and mechanical behavior of the exciter, the diaphragm A_D , the enclosure A_A and passive acoustical elements like port or passive radiator. Employing this knowledge, total sound pressure level SPL(f), state variables (e.g. V_c), transfer functions such as $H_x(f)$ or the electrical impedance $Z_{el}(f)$, as well as efficiency $\eta(f)$ and voltage sensitivity can be easily simulated.

Note that the *LSIM* module only simulates the linear behavior of the system, which is considered valid at small amplitudes. Please see *SIM Simulation* or *SIM-AUR Simulation / Auralization* for nonlinear modeling.

1.2 Input

| Input | The LSIM input is structured into 4 categories: |
|------------|---|
| Parameters | Transducer: Linear transducer parameters (free air) |
| | Enclosure: Type Geometrical properties or lumped parameters |
| | Equalization: High pass filter alignment User defined transfer behavior |
| | Stimulus: Pink noise Typical program material according to IEC 60268-21 User defined spectrum (e.g. music) |
| | The <i>LSIM</i> supports the following enclosure types: |

S58 1 Overview Linear Simulation (LSIM) $V_{\rm b}$ $V_{\rm b}$ $V_{\rm b}$ $S_{\rm d}$ $S_{\rm d}$ S_{d} $S_{\rm d}$ ► $S_{\rm d}$ ŧ. ₿ Sp ₿ Sp $S_{\rm r}$ $l_{\rm p}$ l_{p} V_{i} **Passive Radiator** Bandpass System Baffle **Closed Box** Vented Box Box

1.3 Results

| Linear Transfer Functions | The magnitude and phase frequency responses are calculated between the following state variables • Sound pressure level $L_p(f, r)$ in far field • Displacement (voice coil, passive radiator) • Velocities (voice coil, passive radiator) • Forces in the mechanical system • Volume velocities in the acoustical system in relation to the terminal voltage U_T . The electrical input impedance $Z_e(f)$ is also presented. |
|---------------------------------|--|
| Reference Sensitivity | Voltage sensitivity L(f, r) versus frequency of a sinusoidal stimulus referenced to u_{ref} = 1 V and r_{ref} = 1m. Reference voltage sensitivity L_r for the given broadband stimulus in accordance to IEC 60268-22. |
| Efficiency | Efficiency η(f) versus frequency of a sinusoidal stimulus. Reference efficiency η_r for given broadband stimulus in accordance to IEC 60268-22. |
| Spectra based on Stimulus | For a given broadband stimulus spectrum, the following 1/3rd octave spectra are available: Stimulus (with no filter, band-pass filtered, or band-pass and equalized) Internal state variables (e.g. Displacement for given stimulus) Power (electrical input, acoustical output) |

KLIPPEL Analyzer System

•

 $L_{\rm R}$

Mean efficiency





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| Linear | Simu | lation | (LSIM) | |
|--------|------|--------|--------|--|
| | | | / | |



| • | Pe | Total electrical transducer input power |
|---|----|---|
| • | Pa | Total acoustical output power |

2 Example

| 2.1 Simulat | tion of a vented box system |
|-------------|---|
| Targets | The target of this example is to show a typical workflow on how to use the LSIM for active loudspeaker design. The task for this example is, to use the LSIM for designing a closed box loudspeaker. Therefor the following targets are defined: The desired SPL output is 95 dB. The peak displacement for typical music stimuli should be below x_{max} = 3.5 mm. Due to the targets, the following critical single values should be determined for using typical music signals: Power and voltage consumption (required for selecting an optimal fitting amplifier) Efficiency and voltage-sensitivity Peak displacement |
| User Input | Linear Transducer Parameters: For this example, data of a small midrange speaker was imported from an LPM operation. Immediately after entering all data, the window <i>Table Transducer Parameters</i> shows all loudspeaker parameters. Additional to the entered data some parameters like <i>Cmess</i>, <i>fs</i> and the passband-efficiency of the transducer in free air are calculated and shown. USISM constrained to the sample of the transducer in <i>Constrained Constrained Con</i> |

Linear Simulation (LSIM)

| | Faualization | | | $\overrightarrow{q_a}$ | $q_c = q_1$ | | Vb | | |
|----------------|--|---|---|--|---|---|---|---|---|
| | Target Response | HP-Filter Alignment | | · • | T T | | | | |
| l F | Eilter Tune | Ath order Ruttenworth | | | ļĻ | | | _/ | |
| Ιŀ | for the second s | 140 | | | .¥≶ | i i | | S_{d} | |
| l F | Commente la durta | 140 | | Pbox | | | | | |
| | Compensate Inductan | ce 🛛 | | ¥ | 1 1 | | | | |
| ÷ | Amplifier | | | | | | | | |
| | Enclosure | 1 | | | - | | | | |
| I F | System Type | Closed Box | | Symbol Geometrica | Value U I Parameters of Ac | nit Com oustical Syste | ment em | | |
| | Vb | 1 | | Vb | 1.00 l | Volun | ne of air in end | losure | |
| 11 | Ral | 10000 | | Acoustical F | Parameters Derive | d from Geome | stry | | |
| ÷ | Cone, Radiation, Roor | n | | C _{ab} | 7.13 m | nm ³ /Pa Acous | stical compliant acoustical com | e of air in enclosu pliance of transdu | re cer and enclosure |
| H | | | | a | 1.70 | Syste | m compliance i | ratio = $a = K_{mb} / I$ | Kms |
| Equ | alization | | | Ratc | 188.83 k | Ns/m ^s Total | acoustical resis | stance of transduc | er and enclosure |
| | | | | Mechanical | Parameters Derive | d from Geom | etry | of sis is and some | |
| | Paste Clear | | | Kmt | 6.74 N | /mm Total | mechanical still | fness of transduce | er and enclosure |
| | | | | Derived Par | rameters | | | | |
| | | | | f _c | 168.72 H | z Resor | nance frequenc | y of closed box sy | stem |
| | | ОК Нер | Cluse | V E | | 6 100 | | x system (consid- | cring system load |
| | glectable. At limitation is are entered f | ove 5 kHz nonlinear e entered in section Filte for Target SPL. | effects o er. As tai | rget pe | membra erforman | ne wil ce the | ll beco previ | ome dor ously do | minant. esired 9 |
| | glectable. At limitation is are entered f The window the stimulus enced_After | bove 5 kHz nonlinear of entered in section <i>Filte</i> for <i>Target SPL</i> . stimulus spectrum imr spectrum. All spectra i | effects o er. As tai mediately in this wi | y after ndow | membra erforman defining are relat | ne wil ce the the st ive and | II beco previ imulus d not r | ome dor ously do s param normaliz | ninant. esired 9 eters sł zed or r |
| | glectable. At limitation is are entered f The window the stimulus enced. After well. | pove 5 kHz nonlinear e entered in section <i>Filte</i> for <i>Target SPL</i> . stimulus spectrum imr spectrum. All spectra i clicking on <i>run</i> the b | effects o er. As tai mediately in this wi and-pass | y after ndow s filter | membra erforman defining are relat ed and e | ne wil ce the the st ive and equaliz | II beco previ imulus d not r ed sp | ome dor ously do s param normaliz ectrum | minant. esired 9 eters sh zed or r is visib |
| \15 L | glectable. At limitation is are entered f The window the stimulus enced. After well. | Tutorial Example | effects o er. As tar mediately in this wi and-pass | y after ndow s filter | membra erforman defining are relat ed and e Stir | ne wil ce the the st ive and equaliz nulus | II beco previ imulus d not r ed sp Specti | ome dor ously do s param normaliz ectrum | minant. esired 9 eters sh zed or r is visib |
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| \15 L' | glectable. At limitation is a are entered f The window the stimulus enced. After well. | oove 5 kHz nonlinear e entered in section <i>Filte</i> for <i>Target SPL</i> . stimulus spectrum imr spectrum. All spectra i clicking on <i>run</i> the b Tutorial Example | refects o er. As tai mediately in this wi and-pass | y after ndow | membra erforman defining are relat ed and e Stir Re Stimul | ne wil ce the the st ive and equaliz nulus lative Inp us | II becc previ imulus d not r red sp Specti ut Spectr | ome dor ously do s param normaliz ectrum rum | ninant. esired 9 eters sł zed or r is visib |
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Results The single values listed in table State Variables provides most important information considering the simulated music reproduction: For generating 95 dB SPL output using the desired broadband signal, a voltage of 7.85 V 1. (rms) and current of 1.62 A (rms) are required. The resulting reference efficiency is 0.136 % and the reference voltage-sensitivity is 2. 77.1 dB. 3. Due to the specified crest factor, the amplifier has to provide round about 12.43 W with a peak voltage of 31.27 V. 4. For the specified stimulus a peak displacement of 1.85 mm is expected. Those single values are the basis for defining the amplifier and transducer requirements. Checking the limits defined in the task above reveals, that the desired SPL is possible without of crossing the displacement limit of 3.5 mm. Viewing the curves efficiency and voltage sensitivity versus frequency is useful to check out the limitations of the passive loudspeaker system. The efficiency at lower frequencies decreases rapidly, so pushing frequencies below 100 Hz will be inefficient. Pay attention: Efficiency and voltage-sensitivity are not equal. Efficiency shows the ratio between incoming and outgoing power in percent. Voltage-sensitivity shows the SPL-output at 1 m distance, which is accessible for 1 V at the loudspeaker terminal. η(f) Efficiency L(f) Voltage Sensitivity Acoustical Output Power / Electrical Input Power @Re ference Generator Voltage 1 V and Distance 1 m Effciency (Sinusoidal) Se nsitivity (Sinusoidal) Reference Efficiency (Stimulus) Reference Senst vity (Stimulus) KLIPPEL KLIPPEL 80 Sensitivity/ dB (re 1 Pa/V) 75 Efficiency/ % 70 01 65 Voltage 60 50 100 200 500 1k 2k 50 100 200 500 2k 5k 5k 1k Fr equency / Hz Frequency / Hz For detailed investigations it is useful to view transfer-functions and spectra. It is recommended to check if equalization was reasonable adjusted. Therefor the window H(f,r) Sound Pressure and H(f) Equalizer are relevant: H(f,r) Sound Pressure Transfer Function H(f) Equalizer Transfer Function Magnitude an d Phase of H(f) = Pfar (r,f) / Ug(f) Equalizer Transfer Function Loudspeaker Tar get Phase of Equalizer Transfer Function 180 Loudspeaker + EQ KUPPE 80 20 160 70 ⊣(f) Magnitude / dB (re 20 μPa/V) 0 140 ਛੋਂ Alignment / dB nen 60 120 -20 hase 50 100 -40 deg 80 40 -60 60 30 -80 40 20 50 100 200 500 1k 2k 5k 20 50 100 200 500 1k 2k 5k Fr equency / Hz Fr equency / Hz In the left diagram above the sound pressure transfer function of the passive (black) and active loudspeaker including equalization (blue) is visible. Additional to this the target transfer behavior is shown (red). Comparing the black and blue curve shows a reasonable target transfer behavior. This can be approved by viewing the equalizer transfer function (right diagram

> above). The transfer function contains no excessive damping or boosting. For active loudspeakers displacement is a critical issue. High displacement results undesired distortion and is a limiting factor especially for active control. For investigating this, the displacement transfer-function and displacement spectrum are useful.



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2 Example

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3 Requirements

3.1 Hardware



| License Device | Klippel Dongle or Klippel Analyzer 3 may be used to run this product. | | |
|-------------------|---|--|--|
| 3.2 Software | | | |
| dB-Lab (>210.560) | dB-Lab is the project management software of the KLIPPEL R&D SYSTEM. | | |

4 Parameter

| A.1.1 Electro Dynamic Transducer Parameter Symbol Unit Effective radiation surface d_d cm ² Diameter of round effective radiation surface d_d cm Nominal impedance rated by manufacturer Z_n n Electrical voice-coil resistance at DC R_e Q Voice coil inductance L_e mH Electrical inductance due to eddy current losses L_2 mH Electrical inductance due to eddy current losses R_3 Q Electrical inductance due to eddy current losses L_3 mH Factor of real part in WRIGHT model K Q Exponent of real part in WRIGHT model Krm Q Exponent of imaginary part in WRIGHT model Krm Q Exponent of imaginary part in WRIGHT model Krm Q Exponent of imaginary part in WRIGHT model Krm Q Exponent of real part in WRIGHT model Krm Q Exponent of imaginary part in WRIGHT model Krm Q Exponent of imaginary part in WRIGHT model Krm Q Exponent of real part in WRIGHT model Krm | 4.1 Input | | |
|---|---|--------------------------|-----------------|
| Parameter Symbol Unit Effective radiation surface S_d cm ² Diameter of round effective radiation surface d_d cm Nominal impedance rated by manufacturer Z_n Ω Vicie coil inductance R_e Ω Vicie coil inductance due to eddy current losses L_2 mH Electrical inductance due to eddy current losses L_3 mH Electrical inductance due to eddy current losses L_3 mH Electrical inductance due to eddy current losses R_3 Ω Electrical inductance due to eddy current losses R_3 Ω Exponent in LEACH model n Factor of real part in WRIGHT model K_{rm} Ω Exponent of real part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{sm} Ω Effective instantaneous electrodynamic coupling factor (force factor of the mo- R_{sm} R_{sm} Mechanical resistance of driver supension (inverse of compliance Cam) K_{sm} R_{sm} Mechanical tresistance of driver supp | 4.1.1 Electro Dynamic Transducer | | |
| Effective radiation surface S_d cm^2 Diameter of round effective radiation surface d_d cm Nominal impedance rated by manufacturer Z_n Ω Electrical voice-coil resistance at DC R_e Ω Voice coil inductance L_e mHElectrical inductance due to eddy current losses L_2 mHElectrical inductance due to eddy current losses L_2 mHElectrical inductance due to eddy current losses L_3 mHElectrical inductance due to eddy current losses L_3 mHElectrical inductance due to eddy current losses R_m Ω Electrical inductance due to eddy current losses R_m Ω Exponent in LEACH model n Factor of real part in WRIGHT model K_m Ω Exponent of real part in WRIGHT model K_m Ω Exponent of imaginary part in WRIGHT model K_m Ω Effective instantaneous electrodynamic coupling factor (force factor of the mo- R_m effective instantaneous electrodynamic scoping factor (force factor of the mo- R_m Mechanical resistance of driver suspension losses R_m kg/s Mechanical resistance of driver suspension losses R_m kg/s Mechanical resistance of driver suspension losses R_m kg/s Mechanical resist of driver in free air, considering R_m only (influences R_m g_s Transducer resonance frequency (influences $Rms and M_m)f_sHz4.12Equalizationf_0Hz$ | Parameter | Symbol | Unit |
| Diameter of round effective radiation surface d_d cmNominal impedance rated by manufacturer Z_n Ω Electrical voice-coil resistance at DC R_e Ω Voice coil inductance L_e mHElectric resistance due to eddy current losses R_2 Ω Electric resistance due to eddy current losses R_3 Ω Electric resistance due to eddy current losses R_3 Ω Electric resistance due to eddy current losses R_3 Ω Electric resistance due to eddy current losses L_3 mHFactor in LEACH model n $$ Factor of real part in WRIGHT model K_{rm} Ω Exponent of real part in WRIGHT model E_{rm} $$ Factor of imaginary part in WRIGHT model E_{xm} $$ Exponent of imaginary part in WRIGHT model E_{xm} $$ Effective instantaneous electrodynamic coupling factor (force factor of the mo- tor) defined by the integral of the magnetic flux density B over the voice coil length 1 Bl N/A Mechanical resistance of driver suspension (inverse of compliance C_{m}) K_{ms} N/mm Mechanical astiffness of driver suspension losses R_{ms} kg/s Mechanical mass of driver diaphragm assembly including voice coil and air load M_{ms} gTransducer resonance frequency (influences Rms and M_{m}) f_s HzMechanical Cateor of driver in free air, considering R_{ms} only (influences R_{ms}) Q_{ts} 4.1.2Equalization f_0 < | Effective radiation surface | S _d | cm ² |
| Nominal impedance rated by manufacturer Z_n Ω Electrical voice-coil resistance at DC R_e Ω Voice coil inductance L_e mHElectric resistance due to eddy current losses R_2 Ω Electrical inductance due to eddy current losses L_2 mHElectrical inductance due to eddy current losses L_3 Ω Electrical inductance due to eddy current losses L_3 mHFactor of real part in WRIGHT model K Ω Exponent of real part in WRIGHT model K_{rm} Ω Exponent of real part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{sm} Ω Exponent of imaginary part in WRIGHT model R_{sm} Ω Exponent of imaginary part in WRIGHT model R_{sm} M_{sm} Effective instantaneous electrodynamic coupling factor (force factor of the mo- tor) defined by the integral of the magnetic flux density B over the voice coil length l M_{rms} Mechanical resistance of driver suspension losses R_{ms} kg/s Mechanical resistance of driver suspension losses R_{ms} kg/s Mechanical resistance of driver in free air, considering R_{ms} only (influences R_{ms}) Q_{ts} Atta Equalization f_s HzHigh pass filter alignment: f_0 HzA.Butterworth filter (4 th and 6 th order) f_0 A.Butterworth filter (4 th and 6 th order) f_0 A.Butterworth filter (4 th and 6 th order) f_0 < | Diameter of round effective radiation surface | d_{d} | cm |
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| Voice coil inductance L_e mHElectric resistance due to eddy current losses R_2 Ω Electrical inductance due to eddy current losses L_2 mHElectrical inductance due to eddy current losses L_3 Ω Electrical inductance due to eddy current losses L_3 mHFactor in LEACH model K Ω Exponent in LEACH model K Ω Exponent of real part in WRIGHT model K_{rm} Ω Exponent of real part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{rm} Ω Mechanical stiffness of driver suspension (inverse of compliance C_{ms}) K_{ms} N/mm Mechanical stiffness of driver suspension losses R_{ms} kg/s M_{rm} Mechanical resistance of driver suspension losses R_{ms} kg/s M_{rm} Mechanical Q-factor of driver in free air, considering R_{ms} only (influences R_{ms}) Q_{ts} $$ Alignment | Electrical voice-coil resistance at DC | R _e | Ω |
| Electric resistance due to eddy current losses R_2 Ω Electric resistance due to eddy current losses L_2 mHElectric resistance due to eddy current losses R_3 Ω Electric resistance due to eddy current losses L_3 mHFactor in LEACH model K Ω Factor in LEACH model R Ω Exponent in LEACH model R Ω Exponent of real part in WRIGHT model K_{rm} Ω Exponent of real part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{xm} Ω Exponent of imaginary part in WRIGHT model K_{xm} Ω Effective instantaneous electrodynamic coupling factor (force factor of the mo- tor) defined by the integral of the magnetic flux density B over the voice coil length l Bl N/A Mechanical resistance of driver suspension losses R_{ms} kg/s $M_{mchanical resistance of driver suspension lossesR_{ms}kg/sMechanical Q-factor of driver in free air, considering R_{ms} only (influences R_{ms})Q_{ts}4.12 EqualizationI_{tr}I_{tr}I_{tr}Alignment Type:2. Bessel filter (4^{th} and 6^{th} order)I_{tr}I_{tr}A Butterworth filter (4^{th} and 6^{th} order)I_{tr}$ | Voice coil inductance | Le | mH |
| Electrical inductance due to eddy current losses L_2 mHElectric resistance due to eddy current losses R_3 Ω Electrical inductance due to eddy current losses L_3 mHFactor in LEACH model K Ω Exponent in LEACH model n Factor of real part in WRIGHT model K_{rm} Ω Exponent of real part in WRIGHT model E_{rm} Factor of imaginary part in WRIGHT model E_{rm} Exponent of imaginary part in WRIGHT model E_{xm} Ω Exponent of imaginary part in WRIGHT model E_{xm} Effective instantaneous electrodynamic coupling factor (force factor of the mo- tor) defined by the integral of the magnetic flux density B over the voice coil length 1 Bl N/AMechanical resistance of driver suspension losses R_{ms} kg/sMMechanical resistance of driver suspension losses R_{ms} kg/sMMechanical mass of driver diaphragm assembly including voice coil and air load M_{ms} g4.1.2Equalization f_s HzHigh pass filter alignment:A. Butterworth filter (4 th and 6 th order)3. Chebyshev filter (4 th and 6 th order)4. Butterworth filter (4 th and 6 th order)4. Butterworth filter (4 th and 6 th order)4. Butterworth filter (4 th and 6 th order)4. Chebyshev Con | Electric resistance due to eddy current losses | R_2 | Ω |
| Electric resistance due to eddy current losses R_3 Ω Electrical inductance due to eddy current losses L_3 mHFactor in LEACH model K Ω Exponent in LEACH model n Factor of real part in WRIGHT model K_{rm} Ω Exponent of real part in WRIGHT model K_{rm} Ω Exponent of imaginary part in WRIGHT model K_{xm} Ω Exponent of imaginary part in WRIGHT model K_{xm} Ω Exponent of imaginary part in WRIGHT model K_{xm} Ω Exponent of imaginary part in WRIGHT model K_{xm} Ω Exponent of imaginary part in WRIGHT model K_{xm} Ω Mechanical stiffness of driver suspension (inverse of compliance Cms) R_{ms} R_{g} Mechanical stiffness of driver suspension losses R_{ms} kg/s R/A Mechanical resistance of driver suspension losses R_{ms} kg/s R/A Mechanical Q-factor of driver in free air, considering R_{ms} only (influences R_{ms}) Q_{ts} 4.1.2 Equalization4.1.2 Equalization4.1.2 Equalization4.1.2 Equalization High pass filter alignment: | Electrical inductance due to eddy current losses | L ₂ | mH |
| Electrical inductance due to eddy current losses L_3 mHFactor in LEACH model K Ω Exponent in LEACH model n $$ Factor of real part in WRIGHT model K_{rm} Ω Exponent of real part in WRIGHT model E_{rm} $$ Factor of imaginary part in WRIGHT model E_{rm} $$ Exponent of imaginary part in WRIGHT model E_{xm} Ω Exponent of imaginary part in WRIGHT model E_{xm} $$ Effective instantaneous electrodynamic coupling factor (force factor of the mo- tor) defined by the integral of the magnetic flux density B over the voice coil length 1 Bl N/AMechanical stiffness of driver suspension losses R_{ms} kg/s MMechanical resistance of driver suspension losses R_{ms} kg/s MMechanical ass of driver diaphragm assembly including voice coil and air load M_{ms} ggTransducer resonance frequency (influences Rms and Mms) f_s HzHzMechanical Q-factor of driver in free air, considering R_{ms} only (influences R_{ms}) Q_{ts} 4.1.2 Equalization High pass filter alignment:A. Butterworth filter (4 th and 6 th order)4. Butterworth filter (4 th and 6 th order)4. Butterworth filter (4 th and 6 th order)4. Butterworth filter (4 th and 6 th order) | Electric resistance due to eddy current losses | R ₃ | Ω |
| Factor in LEACH model K Ω Exponent in LEACH model n Factor of real part in WRIGHT model K_{rm} Ω Exponent of real part in WRIGHT model E_{rm} $$ Factor of imaginary part in WRIGHT model K_{xm} Ω Exponent of imaginary part in WRIGHT model K_{xm} Ω Exponent of imaginary part in WRIGHT model K_{xm} Ω Exponent of imaginary part in WRIGHT model E_{xm} Effective instantaneous electrodynamic coupling factor (force factor of the mo- tor) defined by the integral of the magnetic flux density B over the voice coil length I Bl N/AMechanical stiffness of driver suspension (inverse of compliance C_{ms}) K_{ms} N/mmMechanical resistance of driver suspension losses R_{ms} kg/sMechanical resistance of driver suspension losses R_{ms} kg/sMechanical Q-factor of driver in free air, considering R_{ms} only (influences R_{ms}) Q_{ts} 4.1.2 Equalization f_s HzHigh pass filter alignment: $$ $$ A. Biquad filter 1 Biquad filter $$ 3. Chebyshev filter (4 th and 6 th order) S_{0} UnitTarget Cutoff Frequency f_0 HzChebyshev Constant C_{0 $$ Arbitrary target transfer behavior T_{0} HzTarget response as matrix containing frequencies and corresponding levels $$ | Electrical inductance due to eddy current losses | L ₃ | mH |
| Exponent in LEACH modelnFactor of real part in WRIGHT model K_{rm} Ω Exponent of real part in WRIGHT model E_{rm} Factor of imaginary part in WRIGHT model K_{xm} Ω Exponent of imaginary part in WRIGHT model E_{xm} Effective instantaneous electrodynamic coupling factor (force factor of the mo- tor) defined by the integral of the magnetic flux density B over the voice coil length I Bl N/AMechanical stiffness of driver suspension (inverse of compliance Cms) K_{ms} N/mmMechanical resistance of driver suspension losses R_{ms} kg/sMechanical resistance of driver diaphragm assembly including voice coil and air load M_{ms} gTransducer resonance frequency (influences Rms and Mms) f_s HzMechanical Q-factor of driver in free air, considering Rms only (influences Rms) Q_{ts} 4.1.2 Equalization High pass filter alignment:Alignment Type:1. Biquad filter2. Bessel filter (4 th and 6 th order)3. Chebyshev filter (4 th and 6 th order)4. Butterworth filter (4 th and 6 th order)4. Butterworth filter (4 th and 6 th order)4. Chebyshev Constant f_0 HzChebyshev Constant $C_{chebyshev}$ Arbitrary target transfer behavior T_{arget} response as matrix containing frequencies and corresponding levels | Factor in LEACH model | K | Ω |
| Factor of real part in WRIGHT model $K_{\rm rm}$ Ω Exponent of real part in WRIGHT model $E_{\rm rm}$ Factor of imaginary part in WRIGHT model $K_{\rm xm}$ Ω Exponent of imaginary part in WRIGHT model $E_{\rm xm}$ Effective instantaneous electrodynamic coupling factor (force factor of the mo- tor) defined by the integral of the magnetic flux density B over the voice coil length 1 Bl N/AMechanical resistance of driver suspension (inverse of compliance $C_{\rm ms}$) $K_{\rm ms}$ N/mmMechanical resistance of driver suspension losses $R_{\rm ms}$ kg/sMechanical mass of driver diaphragm assembly including voice coil and air load $M_{\rm ms}$ gTransducer resonance frequency (influences Rms and Mms) $f_{\rm s}$ HzMechanical Q-factor of driver in free air, considering Rms only (influences Rms) $Q_{\rm ts}$ 4.1.2EqualizationHigh pass filter alignment:A. Butterworth filter (4th and 6th order)3.Chebyshev filter (4th and 6th order)4.Butterworth filter (4th and 6th order)4.Target Cutoff Frequency f_0 HzChebyshev Constant <t< td=""><td>Exponent in LEACH model</td><td>n</td><td></td></t<> | Exponent in LEACH model | n | |
| Exponent of real part in WRIGHT model $E_{\rm rm}$ $$ Factor of imaginary part in WRIGHT model $K_{\rm xm}$ Ω Exponent of imaginary part in WRIGHT model $E_{\rm xm}$ $$ Effective instantaneous electrodynamic coupling factor (force factor of the motor) defined by the integral of the magnetic flux density B over the voice coil Bl N/Alength I $R_{\rm ms}$ N/A N/A N/A Mechanical stiffness of driver suspension (inverse of compliance $C_{\rm ms}$) $K_{\rm ms}$ N/A Mechanical resistance of driver suspension losses $R_{\rm ms}$ kg/s Mechanical mass of driver diaphragm assembly including voice coil and air load $M_{\rm ms}$ g Transducer resonance frequency (influences Rms and $M_{\rm ms}$) $f_{\rm s}$ HzMechanical Q-factor of driver in free air, considering $R_{\rm ms}$ only (influences $R_{\rm ms}$) $Q_{\rm ts}$ 4.1.2 Equalization $$ $$ $$ $$ High pass filter alignment: $$ $$ $$ A geneter $Symbol$ Unit $$ A geneter f_0 Hz $$ $$ $$ $$ $$ Parameter f_0 Hz $$ $$ $$ $$ $$ Arbitrary target transfer behavior $$ $$ Target response as matrix containing frequencies and corresponding levels $$ | Factor of real part in WRIGHT model | K _{rm} | Ω |
| Factor of imaginary part in WRIGHT model K_{xm} Ω Exponent of imaginary part in WRIGHT model E_{xm} Effective instantaneous electrodynamic coupling factor (force factor of the motor) defined by the integral of the magnetic flux density B over the voice coil length 1BlN/AMechanical stiffness of driver suspension (inverse of compliance C_{ms}) K_{ms} N/mmMechanical resistance of driver suspension losses R_{ms} kg/sMechanical mass of driver diaphragm assembly including voice coil and air load M_{ms} gTransducer resonance frequency (influences Rms and M_{ms}) f_s HzMechanical Q-factor of driver in free air, considering R_{ms} only (influences R_{ms}) Q_{ts} 4.1.2 Equalization High pass filter alignment: 2.Bessel filter (4 th and 6 th order)3. Chebyshev filter (4 th and 6 th order)4. Butterworth filter (4 th and 6 th order) f_0 HzTarget Cutoff Frequency f_0 HzChebyshev Constant $C_{Chebyshev}$ Arbitrary target transfer behavior | Exponent of real part in WRIGHT model | E _{rm} | |
| Exponent of imaginary part in WRIGHT model $E_{\rm xm}$ Effective instantaneous electrodynamic coupling factor (force factor of the mo- tor) defined by the integral of the magnetic flux density B over the voice coil length 1BlN/AMechanical stiffness of driver suspension (inverse of compliance Cms) $K_{\rm ms}$ N/mmMechanical resistance of driver suspension losses $R_{\rm ms}$ kg/sMechanical resistance of driver diaphragm assembly including voice coil and air load $M_{\rm ms}$ gTransducer resonance frequency (influences Rms and Mms) f_s HzMechanical Q-factor of driver in free air, considering Rms only (influences Rms) $Q_{\rm ts}$ 4.1.2EqualizationHigh pass filter alignment: 2.81.Biquad filter2.Bessel filter (4th and 6th order)3.Chebyshev filter (4th and 6th order)4.Butterworth filter (4th and 6th order) <t< td=""><td>Factor of imaginary part in WRIGHT model</td><td>K_{xm}</td><td>Ω</td></t<> | Factor of imaginary part in WRIGHT model | K _{xm} | Ω |
| Effective instantaneous electrodynamic coupling factor (force factor of the mo- tor) defined by the integral of the magnetic flux density B over the voice coil length IBlN/AMechanical stiffness of driver suspension (inverse of compliance Cms)KmsN/mmMechanical resistance of driver suspension lossesRmskg/sMechanical resistance of driver diaphragm assembly including voice coil and air loadMmsgTransducer resonance frequency (influences Rms and Mms)f_sHzMechanical Q-factor of driver in free air, considering Rms only (influences Rms)Qts4.1.2 EqualizationHigh pass filter alignment:Alignment Type: 1. Biquad filter2. Bessel filter (4 th and 6 th order)3. Chebyshev filter (4 th and 6 th order)SymbolUnitTarget Cutoff FrequencyfoHzChebyshev ConstantChebyshev ConstantChebyshev as matrix containing frequencies and corresponding levels | Exponent of imaginary part in WRIGHT model | $E_{\rm xm}$ | |
| Mechanical stiffness of driver suspension (inverse of compliance C_{ms}) K_{ms} N/mmMechanical resistance of driver suspension losses R_{ms} kg/sMechanical mass of driver diaphragm assembly including voice coil and air load M_{ms} gTransducer resonance frequency (influences Rms and Mms) f_s HzMechanical Q-factor of driver in free air, considering Rms only (influences Rms) Q_{ts} 4.1.2 Equalization High pass filter alignment:Alignment Type:1. Biquad filter2. Bessel filter (4^{th} and 6^{th} order)3. Chebyshev filter (4^{th} and 6^{th} order)4. Butterworth filter (4^{th} and 6^{th} order)4. Butterworth filter (4^{th} and 6^{th} order)7arget Cutoff Frequency f_0 HzChebyshev Constant $C_{Chebyshev}$ Arbitrary target transfer behaviorTarget response as matrix containing frequencies and corresponding levels | Effective instantaneous electrodynamic coupling factor (force factor of the mo- tor) defined by the integral of the magnetic flux density B over the voice coil length l | Bl | N/A |
| Mechanical resistance of driver suspension losses R_{ms} kg/sMechanical mass of driver diaphragm assembly including voice coil and air load M_{ms} gTransducer resonance frequency (influences Rms and Mms) f_s HzMechanical Q-factor of driver in free air, considering Rms only (influences Rms) Q_{ts} 4.1.2 Equalization High pass filter alignment:Alignment Type:1. Biquad filter2. Bessel filter (4 th and 6 th order)3. Chebyshev filter (4 th and 6 th order)4. Butterworth filter (4 th and 6 th order)4. Butterworth filter (4 th and 6 th order)fo4. Butterworth filter (4 th and 6 th order)5. Chebyshev Constant f_0 HzArbitrary target transfer behaviorTarget response as matrix containing frequencies and corresponding levels | Mechanical stiffness of driver suspension (inverse of compliance C _{ms}) | K _{ms} | N/mm |
| Mechanical mass of driver diaphragm assembly including voice coil and air load M_{ms} gTransducer resonance frequency (influences Rms and Mms) f_s HzMechanical Q-factor of driver in free air, considering Rms only (influences Rms) Q_{ts} 4.1.2 Equalization High pass filter alignment:Alignment Type:1. Biquad filter2. Bessel filter (4th and 6th order)3. Chebyshev filter (4th and 6th order)4. Butterworth filter (4th and 6th order)4. Butterworth filter (4th and 6th order)Chebyshev Constant f_0 HzArbitrary target transfer behaviorTarget response as matrix containing frequencies and corresponding levels | Mechanical resistance of driver suspension losses | R _{ms} | kg/s |
| Transducer resonance frequency (influences Rms and Mms) $f_{\rm s}$ HzMechanical Q-factor of driver in free air, considering Rms only (influences Rms) $Q_{\rm ts}$ 4.1.2 EqualizationHigh pass filter alignment: Alignment Type: 1. Biquad filter 2. Bessel filter (4 th and 6 th order) 3. Chebyshev filter (4 th and 6 th order) 4. Butterworth filter (4 th and 6 th order) 4. Butterworth filter (4 th and 6 th order) 4. Butterworth filter (4 th and 6 th order) | Mechanical mass of driver diaphragm assembly including voice coil and air load | M _{ms} | g |
| Mechanical Q-factor of driver in free air, considering Rms only (influences Rms) Qts 4.1.2 Equalization High pass filter alignment: Alignment Type: . . . 1. Biquad filter . . . 2. Bessel filter (4 th and 6 th order) . . . 3. Chebyshev filter (4 th and 6 th order) . . . 4. Butterworth filter (4 th and 6 th order) . . . 7arget Cutoff Frequency f_0 Hz Chebyshev Constant . . Arbitrary target transfer behavior Target response as matrix containing frequencies and corresponding levels | Transducer resonance frequency (influences Rms and Mms) | f _s | Hz |
| 4.1.2 Equalization High pass filter alignment: Alignment Type: 1. Biquad filter 2. Bessel filter (4 th and 6 th order) 3. Chebyshev filter (4 th and 6 th order) 4. Butterworth filter (4 th and 6 th order) 4. Butterworth filter (4 th and 6 th order) 7 arget Cutoff Frequency f_0 Farget Cutoff Frequency f_0 Chebyshev Constant C _{Chebyshev} Arbitrary target transfer behavior Target response as matrix containing frequencies and corresponding levels | Mechanical Q-factor of driver in free air, considering R_{ms} only (influences R_{ms}) | $Q_{\rm ts}$ | |
| High pass filter alignment: Alignment Type: 1. Biquad filter 2. Bessel filter (4 th and 6 th order) 3. Chebyshev filter (4 th and 6 th order) 4. Butterworth filter (4 th and 6 th order) 7 Target Cutoff Frequency Symbol Target Cutoff Frequency f ₀ Hz Chebyshev Constant Arbitrary target transfer behavior Target response as matrix containing frequencies and corresponding levels | 4.1.2 Equalization | | |
| Alignment Type: 1. Biquad filter 2. Bessel filter (4 th and 6 th order) 3. Chebyshev filter (4 th and 6 th order) 4. Butterworth filter (4 th and 6 th order) 7. Betterworth filter (4 th and 6 th order) 6. Butterworth filter (4 th and 6 th order) 7. Butterworth filter (4 th and 6 th order) 7. Butterworth filter (4 th and 6 th order) 6. Butterworth filter (4 th and 6 th order) 7 Target Cutoff Frequency Symbol 9. Chebyshev Constant f_0 1. Arbitrary target transfer behavior $C_{Chebyshev}$ 1. Target response as matrix containing frequencies and corresponding levels V | High pass filter alignment: | | |
| Parameter Symbol Unit Target Cutoff Frequency f_0 Hz Chebyshev Constant $C_{Chebyshev}$ Arbitrary target transfer behavior Target response as matrix containing frequencies and corresponding levels | Alignment Type: 1. Biquad filter 2. Bessel filter (4th and 6th order) 3. Chebyshev filter (4th and 6th order) 4. Butterworth filter (4th and 6th order) | | |
| Target Cutoff Frequency f_0 HzChebyshev Constant $\mathcal{C}_{Chebyshev}$ Arbitrary target transfer behaviorTarget response as matrix containing frequencies and corresponding levels | Parameter | Symbol | Unit |
| Chebyshev Constant $C_{Chebyshev}$ Arbitrary target transfer behaviorTarget response as matrix containing frequencies and corresponding levels | Target Cutoff Frequency | f_0 | Hz |
| Arbitrary target transfer behavior Target response as matrix containing frequencies and corresponding levels | Chebyshev Constant | $C_{\mathrm{Chebyshev}}$ | |
| Target response as matrix containing frequencies and corresponding levels | Arbitrary target transfer behavior | | |
| | Target response as matrix containing frequencies and corresponding levels | | |

| 4.1.3 Amplifier | | |
|--|----------------|------|
| Parameter | Symbol | Unit |
| Output-resistance of amplifier output including cables | R _g | Ω |

| 4.1.4 Stimulus | | | | | |
|---|-----------------------------------|--------------------|--|--|--|
| Type of input signal: | | | | | |
| 1. Pink noise | | | | | |
| 2. Typical program (IEC 60268-21) | 2. Typical program (IEC 60268-21) | | | | |
| 3. External spectrum | | | | | |
| Bandpass: | | | | | |
| 1. Ideal (rectangle) | | | | | |
| 2. Butterworth | | | | | |
| Parameter | Symbol | Unit | | | |
| Cutoff frequency of the high pass filter | fend | Hz | | | |
| Slope of high pass filter | $m_{\mu\nu}$ | dB | | | |
| Cutoff frequency of the Low pass filter | f _{cLP} | Hz | | | |
| Slope of low pass filter | | dB | | | |
| Crest factor | CF | dB | | | |
| Difference between crest factor for voltage and current signal and crest factor | | - ub | | | |
| for displacement signal | ΔCF | dB | | | |
| 4.1.5 Enclosure | | | | | |
| Enclosure type: | | | | | |
| 1. Baffle | | | | | |
| 2 Closed box | | | | | |
| Vented box (with slit or tube-shaped vent) | | | | | |
| 4 Box with passive radiator | | | | | |
| 5 Bandnass system (with slit or tube-shaned vent) | | | | | |
| | | | | | |
| Parameter | Symbol | Unit | | | |
| Geometrical parameters: | | • | | | |
| Volume of air in enclosure | V _b | <u> </u> | | | |
| Surface area of port | S _p | cm² | | | |
| Diameter of port | $d_{ m p}$ | cm | | | |
| Length of port | lp | cm | | | |
| Width of surface area of port | w _p | cm | | | |
| Height of surface area of port | $h_{ m p}$ | cm | | | |
| Effective projected surface area of passive radiator diaphragm | S _r | cm ² | | | |
| Diameter of round effective projected surface area of passive radiator dia- | | am | | | |
| phragm | $u_{\rm r}$ | CIII | | | |
| Volume of air in front enclosure | $V_{\rm f}$ | 1 | | | |
| Lumped parameters: | | | | | |
| Acoustic resistance of losses due to leakage | R _{al} | kNs/m ⁵ | | | |
| Acoustic mass of port including air load | $R_{\rm ap}$ | kNs/m ³ | | | |
| Acoustic resistance of port losses | M _{ap} | kg/m ⁴ | | | |
| Mechanical mass of passive radiator diaphragm including voice coil and air load | M _{mr} | g | | | |
| Mechanical stiffness of passive radiator suspension (inverse of compliance C_{mr}) | K _{mr} | N/mm | | | |
| Mechanical resistance of passive radiator suspension losses | R _{mr} | kg/s | | | |
| Derived parameters: | Derived parameters: | | | | |
| Q-factor of acoustic system at fb considering leakage losses | Q_1 | | | | |
| Resonance frequency of enclosure-port system | $f_{\rm b}$ | Hz | | | |
| Q-factor considering port losses | Q _n | | | | |
| Resonance frequency of enclosure-port system $f_{\rm f}$ | | | | | |
| 4.1.6 Room and Radiation | | | | | |
| Radiation into half and full space: 2π or 4π (anechoic, piston) | | 1 | | | |
| Parameter | Symbol | Unit | | | |
| Distance to radiation point in far field | $r_{\rm ref}$ | m | | | |

| 4.2 Results | | | | |
|--|--------------------------------|--------------------|--|--|
| 4.2.1 Electro-dynamical Transducer | | | | |
| Parameter | Symbol | Unit | | |
| Derived parameters: | | | | |
| Transducer resonance frequency (influences $R_{\rm ms}$ and $M_{\rm ms}$) | fs | Hz | | |
| Mechanical Q-factor of driver in free air, considering R_{ms} only | 0 _{ms} | | | |
| Electrical Q-factor of driver in free air, considering R_e only | O_{es} | | | |
| Mechanical Q-factor of driver in free air, considering Rms only (influences R_{ms}) | Q_{ts} | | | |
| Equivalent air volume of driver suspension | Vas | 1 | | |
| Efficiency and Sensitivity: | 45 | | | |
| Passband efficiency of driver operated in baffle | $\eta_{ m Pb}$ | % | | |
| Passband sensitivity of driver operated in baffle with reference voltage uref and | 110 | 10 | | |
| reference distance r _{ref} defined in ppg. | $L_{\rm Pb}$ | dB | | |
| | | | | |
| 4.2.2 Enclosure | | | | |
| Parameter | Symbol | Unit | | |
| Lumped parameters: | | | | |
| Acoustical compliance of air in enclosure | \mathcal{C}_{ab} | m ³ /Pa | | |
| Mechanical stiffness of air in enclosure | K _{mb} | N/mm | | |
| Acoustical compliance of air in front enclosure | $C_{\rm f}$ | m ³ /Pa | | |
| Total acoustical compliance of transducer and enclosure | C _{at} | m ³ /Pa | | |
| Total mechanical stiffness of transducer and enclosure | K _{mt} | N/mm | | |
| System compliance ratio | α | | | |
| Derived parameters: | | | | |
| Resonance frequency of the closed box system | fc | Hz | | |
| Passive-Radiator resonance frequency (free air) | fn | Hz | | |
| Mechanical Q-factor of passive radiator in free air, considering R _{mr} only | $Q_{\rm mn}$ | | | |
| Total Q-factor considering all acoustical losses | $\frac{O_{\rm h}}{O_{\rm h}}$ | | | |
| Q-factor of the closed box system (considering system load) | $\frac{c_{\rm b}}{Q_{\rm tc}}$ | | | |
| 4.2.3 State Variables and Further Characteristics (depending on stimulus) | | | | |
| Parameter | Symbol | Unit | | |
| Reference Voltage-Sensitivity of selected stimulus for $r_{ref} = 1 \text{ m}$ and $u_{ref} = 1 \text{ V}$ | $L_{\rm R}$ | dB | | |
| according to IEC 60268-22 | | | | |
| Reference efficiency for selected stimulus according to IEC 60268-22 | $\eta_{ m R}$ | % | | |
| Far field SPL at distance r _{ref} for stimulus | $L_{p_{far}}$ | dB | | |
| Terminal voltage (rms) for stimulus | $U_{\mathrm{T}_{rms}}$ | V | | |
| Generator voltage (rms) for stimulus | U _{Grms} | V | | |
| Terminal voltage (peak) for stimulus | UTnoak | V | | |
| Generator voltage (peak) for stimulus | U _{Gpeak} | V | | |
| Input current (rms) for stimulus | ITrms | Α | | |
| Input current (peak) for stimulus | I _{Tuesh} | Α | | |
| Voice coil displacement (rms) for stimulus | X - | mm | | |
| Voice coil displacement (neak) for stimulus | | | | |
| Voice coil volocity (rms) for stimulus | | | | |
| SDL in roar air volume for stimulus | v _{crms} | | | |
| SPL in rear air volume for stimulus | $p_{ m box}$ | uв | | |
| 4.2.4 Transfer functions | . | | | |
| Function | Symbol | Unit | | |
| Voltage Sensitivity | L(f) | dB | | |
| Efficiency | $\eta(f)$ | % | | |
| Electrical Impedance: | | | | |

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| Total electrical impedance | $Z_{o}(f)$ | Ω |
|---|----------------------------|-----|
| Back EMF | Blv/u_{a} | Ω |
| DC-Resistance of the transducer and the amplifier output resistance | $R_{o} + R_{a}$ | Ω |
| Voice coil impedance | $Z_{al}(f)$ | .0. |
| Far Field Sound Pressure: | 26107 | |
| Total Sound Pressure | $H_{\rm nc}(f,r)$ | dB |
| Contribution from port | $H_{\rm p}(f,r)$ | dB |
| Target sound pressure | $H_{t}(f,r)$ | dB |
| Total active system (with equalization) | $H_{\text{total}}(f,r)$ | dB |
| Displacement divided by generator voltage: | | |
| Voice coil | $x_{\rm c}(f)/u_{\rm g}$ | dB |
| Passive radiator | $x_{\rm r}(f)/u_{\rm g}$ | dB |
| Velocity divided by generator voltage: | 8 | |
| Voice coil | $v_{\rm c}(f)/u_{\rm g}$ | dB |
| Passive radiator | $v_{\rm r/p}(f)/u_{\rm g}$ | dB |
| Force divided by generator voltage: | -// • // | |
| At the motor | $F_{\rm c}(f)/u_{\rm g}$ | dB |
| At M _{ms} | $F_{\rm Mms}(f)/u_{\rm g}$ | dB |
| At R _{ms} | $F_{\rm Rms}(f)/u_{\rm g}$ | dB |
| At C_{ms} | $F_{\rm Cms}(f)/u_{\rm g}$ | dB |
| Into the acoustical system | $F_{\rm L}(f)/u_{\rm g}$ | dB |
| Volume velocity divided by generator voltage: | 2411 8 | |
| From S _d | $q_{\rm Sd}(f)/u_{\rm g}$ | dB |
| Into C _{ab} | $q_{\rm c}(f)/u_{\rm g}$ | dB |
| Into C_f | $q_{\rm f}(f)/u_{\sigma}$ | dB |
| Into R_{al} | $q_1(f)/u_{\sigma}$ | dB |
| Into port/passive radiator | $q_{\rm p}(f)/u_{\rm g}$ | dB |
| Amplifier transfer function (voltage drop) | $u_{t}(f)/u_{q}$ | dB |
| Prefilter transfer function (Equalizer) | $H_{equ}(f)$ | dB |
| Stimulus Spectrum: | cqu o y | |
| Relative input spectrum | $G_{\rm w}(f)$ | dB |
| Aligned input spectrum | $G_{eq}(f)$ | dB |
| Voltage Spectrum: | | |
| Terminal voltage | u _t | dB |
| Amplifier output voltage without load | $u_{\rm g}$ | dB |
| Power Spectrum: | · • • | |
| Electrical generator output power | Pe | dB |
| Acoustical output power | Pa | dB |
| Power dissipation in amplifier | $P_{R_{g}}$ | dB |
| Spectrum of the sound pressure level | $L_{p_{far}}$ | dB |
| State Spectrum: | - 144 | |
| Voice Coil Displacement | $L_{x_{coil}}$ | dB |
| Voice Coil Velocity | $L_{v_{coil}}$ | dB |
| Voice Coil Force | $L_{F_{coil}}$ | dB |
| Radiated Volume Velocity | L_{q_a} | dB |

5 References

| 5.1 | Related Modules | LPM Linear Parameter Measurement |
|-----|-----------------|-----------------------------------|
| | | SIM Simulation |
| | | SIM-AUR Simulation / Auralization |

| 5.2 | Manuals | LSIM Manual, as provided with dB-Lab 210.560 or higher |
|-----|-----------------------|---|
| 5.3 | Related Papers | Wolfgang Klippel: "Green Speaker Design (Part 1: Optimal Use of System Re- sources)", 2019, Klippel GmbH |
| | | Wolfgang Klippel: " <u>Green Speaker Design (Part 2: Optimal Use of Transducer</u> <u>Resources)</u> ", 2019, Klippel GmbH |
| | | R. H. Small: "Closed-Box Loudspeaker Systems", 2006, School of electrical Engineering, The University of Sydney, Australia |

Find explanations for symbols at: http://www.klippel.de/know-how/literature.html Last updated: May 25, 2021

